



The Determination of Emissivity T A Stationary Method

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I. INTRODUCTION

A review of methods for measuring the integral radiation characteristics of materials, carried out in the article and shows that there are the following tasks for creating simplified measurement methods [1-2].

II. MATERIALS AND METHODS

In the widely used "radiation method", it is necessary to measure three parameters - temperature and radiation flux density, as well as convective heat loss (if convective heat loss is excluded, as can be seen, it is necessary to measure two quantities). To exclude convective heat losses and the need to measure the radiation flux, a simplified non-stationary method for determining the integral emissivity (ϵ_T) was proposed in [3-4]. Where it was proposed to exclude convective heat losses by conducting an experiment simultaneously with two samples - one with a known ϵ_T , is the second working. Both methods were based on the fact that at the same temperatures and for the same temperature differences, both samples have the same convective heat loss. However, as is known for non-stationary methods, samples with the same dimensions and known thermo-physical characteristics (heat capacity, mass) are required, and it is also necessary to measure time. Both of these methods were proposed, but were not implemented, and the error assessment of these methods was not carried out.

The analysis showed that it is possible to develop the methods proposed above and to develop on this basis a stationary method - measuring the emissivity by equilibrium temperatures. A schematic of our stationary method is shown in Fig. 1. Its peculiarity is as follows. Two thin flat plates are taken, from one working material with a thickness of about 1-5 mm, and one of them is covered with soot. These samples, working and "black" are heated using a radiation source. The equilibrium temperatures of the samples are measured. Further, given that the temperature drops in the samples are small, less than 0.1 degrees, and also neglecting heat loss through the sides of the plates and heat loss through the support legs, we can write the following balance equations.

$$2 \cdot \epsilon_1 \cdot \sigma \cdot T^4 + 2 \cdot \alpha_{K1} \cdot (T - T_0) = \alpha_1 \cdot E_C + 2 \cdot \epsilon_1 \cdot \sigma \cdot T_{\Pi}^4 \quad (1)$$

$$2 \cdot \epsilon_2 \cdot \sigma \cdot T_a^4 + 2 \cdot \alpha_{K2} \cdot (T_a - T_0) = \alpha_2 \cdot E_C + 2 \cdot \epsilon_1 \cdot \sigma \cdot T_{\Pi}^4 \quad (2)$$

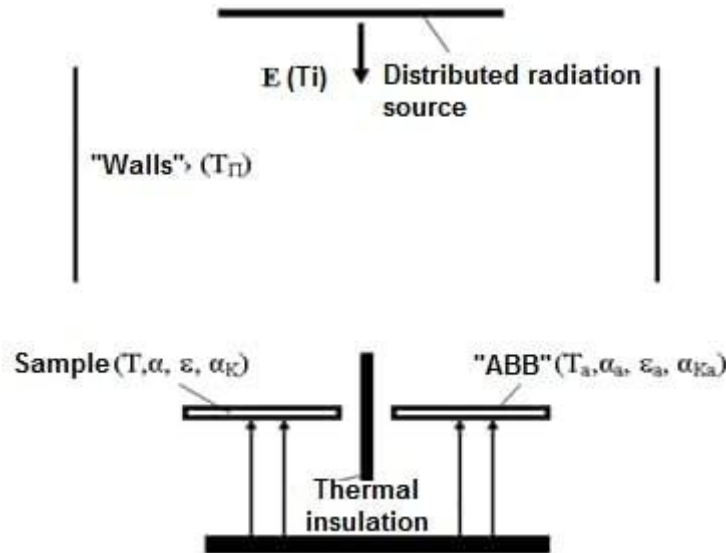


Figure 1. Scheme for determining ϵ_T from equilibrium temperatures.

At source temperatures close to heating temperatures, it can also be assumed that

$$\alpha_1 = \epsilon_1 \text{ and } \alpha_2 = \epsilon_2 \tag{3}$$

In these equations, the temperatures of the sample and the "ABB", the temperatures of the walls (measured), the ambient air temperature are known, the density of the incident radiation E is also measured, and the unknowns are the convective heat transfer coefficients α_{K1}, α_{K2} .

From equation (2) we determine the coefficient of convective heat transfer α_{K2} . Let's find the connection between α_{K2} and α_{K1} . In general, the nature of convective heat transfer on both samples is the same (the same conditions), so the differences between them are due to the difference in the temperatures of the working sample and the "ABB". The temperature dependence of the convective heat transfer coefficient for flat plates cooled from above is shown in Fig. 2.

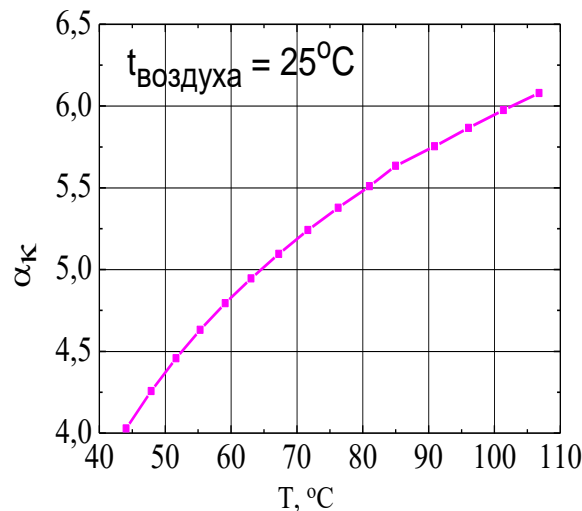


Fig. 2. Dependence of the coefficient of natural convective heat transfer from the temperature of a flat plate cooled from above.

As shown by preliminary experiments, the difference between the temperatures of the working sample and the "ABB" is at the level of 10-15, while as follows from Fig.2. The difference between α_{K2} and α_{K1} does not exceed 5%. It can be assumed that $\alpha_{K1} = \alpha_{K2}$. Then, determining α_{K2} from (2) and substituting instead of α_{K1} from (1), we determine the emissivity of the sample. Note that in the case of forced convection, the heat transfer coefficient does not depend on the temperature of the samples, but depends only on the temperature of the ambient air.

Let us estimate the main components of the relative error of the method δ . They are made up of the following random errors:

The error in the assumption of low heat loss from the side surface is $-\delta_S = 100\% \cdot (S_F / (2 \cdot S)) = 100\%$, which at $r_S = 30\text{mm}$ and $h = 2\text{mm}$ is equal to $\delta_S = 100\% (h / r_S) = 100 \cdot 0.033 = 3.3\%$;

The error in determining the temperatures of the sample and "ABB" by thermocouples $-\delta_{TO} = 100\% \cdot (0.5/90) = 0.6\%$;

The error in determining the density of the incident flow $-\delta_{\Pi A \Pi} = 5\%$ (according to the passport);

The error in determining the ambient air temperature $-\delta_B = 100\% \cdot (0.2/25) = 0.8\%$;

The error in the assumption that $\alpha_{K1} = \alpha_{K2}$ in the case of natural convection is $-\delta_{\alpha K} = 5\%$, in the case of forced convection $\delta_{\alpha K} = 0\%$;

The error in determining the temperature of the "walls" $-\delta_{CT} = 100\% \cdot (0.5/30) = 1.7\%$;

The total relative error δ will be determined by the formula [5].

$$\delta = (\delta_S^2 + \delta_{TO}^2 + \delta_{T A \Pi T}^2 + \delta_{\Pi A \Pi}^2 + \delta_B^2 + \delta_{\alpha K}^2 + \delta_{CT}^2)^{0.5} = 8\% \quad (4)$$

Consequently, for almost all bodies it is possible to determine the integral emissivity and the selectivity parameter to solar radiation from its equilibrium temperature, and since the equilibrium temperatures of the gray and black bodies are the same, a blackened working sample can be used as a gray body.

Figure 3 shows a diagram of an installation for measuring emissivity, where: 1 - sample $-\varepsilon_T$, which must be determined; 2 - "black" sample with known ε_T ; 3 - type XA thermocouples; 4 - switch; 5 - voltmeter; 6 - thermometer; 7 - sensor for measuring the radiation flux, type FOA 022, 8 - distributed radiation source, 9 - standard actinometer for measuring direct solar radiation with an automatic tracking system for the Sun.

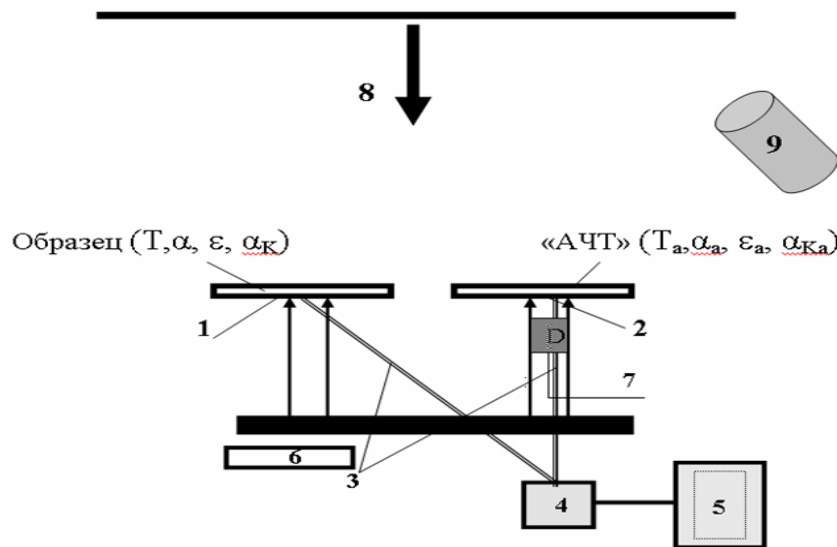


Fig. 3. Installation diagram for determining the emissivity

When developing the experimental setup, the requirements of the developed technique were taken into account (see [1]) - ensuring a uniform flux on the samples, the possibility of excluding the influence of residual radiation from the source on the samples after it is turned off (when the samples reach an equilibrium temperature), this was the reason for the use of quartz halogen lamps, the maximum possible elimination of heat leakage through the sample holders (needle supports for samples), as well as ensuring the possibility of fulfilling the conditions of equality of the emissivity and absorption capacity of the material during measurements (due to a change in the temperature of the radiation source). A general view of the experimental setup for measuring the emissivity ε_T of various opaque materials is shown in Fig. 4.

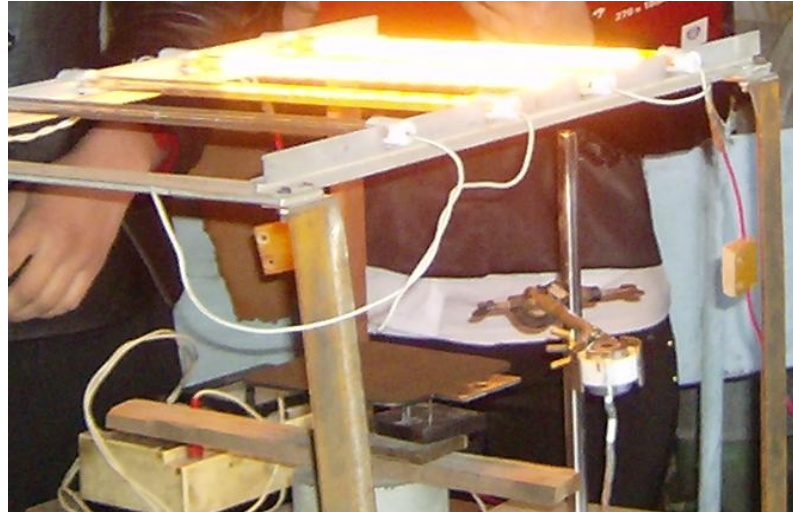


Fig. 4. An experimental setup for determining the emissivity.

The procedure for the experimental determination of ε_T by the method developed above was as follows. Two plates (working and "black") with thermocouples preliminarily fixed on their lower side are placed on the support legs of the installation. Then the source is turned on and when the samples reach equilibrium temperatures, the source is turned off. Based on the data obtained, convective heat transfer coefficient for a black sample.

$$\alpha_K = [\varepsilon_{T,A\check{T}}*(E_C - 2*\sigma*(T_{A\check{T}}^4 - T_B^4))] / [2*(T_{A\check{T}} - T_B)] \quad (5)$$

Working material emissivity

$$\varepsilon_{T,O\check{B}P} = [2*\alpha_K*(T_{O\check{B}P} - T_0)] / (E_C - 2*\sigma*(T_{O\check{B}P}^4 - T_B^4)) \quad (6)$$

III. CONCLUSION

To check the experimental setup and refine the technique, we first measured ε_T of various materials with known ε_T copper and aluminum plates. It was obtained for a copper plate $\varepsilon_T = 0.44$, and for an aluminum plate $\varepsilon_T = 0.32$. These data, with an error of 8% to 15%, agree with those given in the literature [2-4]. Large deviations of the ε_T values were due to the fact that real plates were used in the experiment, and which were not specially processed. Further, after working out the method, the emissivity of some widely used materials was investigated - paints (white, red, green, blue, black and gray enamels) and building materials (gypsum, chamotte, limestone). The purpose of these studies was to determine in the subsequent their absorbency to solar radiation.

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